Fracture Mechanics, Damage and Fatigue Linear Elastic Fracture Mechanics - Asymptotic Solution

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Fracture Mechanics - LEFM - Asymptotic Solution

Linear Elastic Fracture Mechanics (LEFM)



Cracked body

- Where do the asymptotic solutions come form?





LEFM assumptions

- Balance of body *B*
 - Momenta balance
 - Linear
 - Angular
 - Boundary conditions
 - Neumann
 - Dirichlet



• Small deformations with linear elastic, homogeneous & isotropic material

$$- \text{ (Small) Strain tensor } \boldsymbol{\varepsilon} = \frac{1}{2} \left(\boldsymbol{\nabla} \otimes \boldsymbol{u} + \boldsymbol{u} \otimes \boldsymbol{\nabla} \right), \text{ or } \begin{cases} \boldsymbol{\varepsilon}_{ij} = \frac{1}{2} \left(\frac{\partial}{\partial \boldsymbol{x}_i} \boldsymbol{u}_j + \frac{\partial}{\partial \boldsymbol{x}_j} \boldsymbol{u}_i \right) \\ \boldsymbol{\varepsilon}_{ij} = \frac{1}{2} \left(\boldsymbol{u}_{j,i} + \boldsymbol{u}_{i,j} \right) \end{cases}$$

– Hooke's law
$$oldsymbol{\sigma}=\mathcal{H}:oldsymbol{arepsilon}$$
 , or $oldsymbol{\sigma}_{ij}=\mathcal{H}_{ijkl}oldsymbol{arepsilon}_{kl}$

with
$$\mathcal{H}_{ijkl} = \underbrace{\frac{E\nu}{(1+\nu)(1-2\nu)}}_{\lambda = K-2\mu/3} \delta_{ij}\delta_{kl} + \underbrace{\frac{E}{1+\nu}}_{2\mu} \left(\frac{1}{2}\delta_{ik}\delta_{jl} + \frac{1}{2}\delta_{il}\delta_{jk}\right)$$

e law $\varepsilon = \mathcal{G}: \sigma$

Inverse law

with

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 $\mathcal{G}_{ijkl} = \frac{1+\nu}{E} \left(\frac{1}{2} \delta_{ik} \delta_{jl} + \frac{1}{2} \delta_{il} \delta_{jk} \right) - \frac{\nu}{E} \delta_{ij} \delta_{kl}$





- Static _
- 2D &
 - Plane $\sigma \implies \sigma_{xz} = \sigma_{yz} = \sigma_{zz} = 0$

• Or plane
$$\varepsilon \implies \begin{cases} \varepsilon_{xz} = \varepsilon_{yz} = \varepsilon_{zz} = 0\\ \frac{\partial}{\partial z} = 0 \end{cases}$$







Static & 2D assumptions: Plane- ε case

- Plane
$$\varepsilon$$
:
• $\varepsilon_{xz} = \varepsilon_{yz} = \varepsilon_{zz} = 0$ & $\frac{\partial}{\partial z} = 0$

- Hooke's law
• General
$$\begin{cases} \sigma_{ij} = \mathcal{H}_{ijkl}\varepsilon_{kl} \\ \mathcal{H}_{ijkl} = \frac{E\nu}{(1+\nu)(1-2\nu)}\delta_{ij}\delta_{kl} + \frac{E}{1+\nu}\left(\frac{1}{2}\delta_{ik}\delta_{jl} + \frac{1}{2}\delta_{il}\delta_{jk}\right) \\ \bullet \text{ Plane }\varepsilon \\ = \int_{0}^{\infty} \sigma_{xx} = \frac{E\nu}{(1+\nu)(1-2\nu)}\delta_{xx}\varepsilon_{kk} + \frac{\varepsilon}{1+\nu}\varepsilon_{xx} + \varepsilon_{yy} + \varepsilon_{zz} = \varepsilon_{xx} + \varepsilon_{yy} = \varepsilon_{\alpha\alpha} \\ \sigma_{xy} = \frac{E\nu}{(1+\nu)(1-2\nu)}\delta_{xx}\varepsilon_{kk} + \frac{E}{1+\nu}\varepsilon_{xx} \\ \sigma_{\alpha\beta} = \frac{E\nu}{(1+\nu)(1-2\nu)}\delta_{\alpha\beta}\varepsilon_{\gamma\gamma} + \frac{E}{1+\nu}\varepsilon_{\alpha\beta} \end{cases}$$
Greek subscripts for x or y Roman subscripts for x, y or Roman s

Greek subscripts for *x* or *y* Roman subscripts for x, y or z

- N.B.:
$$\sigma_{\gamma\gamma} = \frac{E\nu}{(1+\nu)(1-2\nu)} \underbrace{\delta_{\gamma\gamma} \varepsilon_{\delta\delta}}_{2\varepsilon_{\delta\delta}} + \frac{E}{1+\nu} \varepsilon_{\gamma\gamma} = \frac{E}{(1+\nu)(1-2\nu)} \varepsilon_{\gamma\gamma}$$

 $2\varepsilon_{\delta\delta} = 2\varepsilon_{xx} + 2\varepsilon_{yy} = 2\varepsilon_{\gamma\gamma}$



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• Static & 2D assumptions: Plane- ε case (2)

- Plane
$$\varepsilon$$
:
• $\varepsilon_{xz} = \varepsilon_{yz} = \varepsilon_{zz} = 0$ & $\frac{\partial}{\partial z} = 0$

- Hooke's law
• General
$$\begin{cases} \boldsymbol{\sigma}_{ij} = \mathcal{H}_{ijkl} \boldsymbol{\varepsilon}_{kl} \\ \mathcal{H}_{ijkl} = \frac{E\nu}{(1+\nu)(1-2\nu)} \delta_{ij} \delta_{kl} + \frac{E}{1+\nu} \left(\frac{1}{2} \delta_{ik} \delta_{jl} + \frac{1}{2} \delta_{il} \delta_{jk}\right) \end{cases}$$

• Plane ε

$$\int \sigma_{\alpha\beta} = \frac{E\nu}{(1+\nu)(1-2\nu)} \delta_{\alpha\beta} \varepsilon_{\gamma\gamma} + \frac{E}{1+\nu} \varepsilon_{\alpha\beta}$$

$$\sigma_{\gamma\gamma} = \frac{E}{(1+\nu)(1-2\nu)} \varepsilon_{\gamma\gamma}$$

$$\sigma_{zz} = \frac{E\nu}{(1+\nu)(1-2\nu)} \delta_{zz} \varepsilon_{kk} + \frac{E}{1+\nu} \varepsilon_{zz}$$

$$\varepsilon_{\delta\delta} = \varepsilon_{xx} + \varepsilon_{yy} = \varepsilon_{\gamma\gamma}$$

Greek subscripts for *x* or *y* Roman subscripts for *x*, *y* or *z*

$$\implies \boldsymbol{\sigma}_{zz} = \frac{E\nu}{(1+\nu)(1-2\nu)} \boldsymbol{\varepsilon}_{\gamma\gamma} = \nu \boldsymbol{\sigma}_{\gamma\gamma}$$





- Static & 2D assumptions: Plane- ε case (3)
 - Plane ε : • $\varepsilon_{xz} = \varepsilon_{yz} = \varepsilon_{zz} = 0$ & $\frac{\partial}{\partial z} = 0$

•
$$\sigma_{zz} = \frac{E\nu}{(1+\nu)(1-2\nu)} \varepsilon_{\gamma\gamma} = \nu \sigma_{\gamma\gamma}$$

Hooke's law

$$\varepsilon = \mathcal{G} : \boldsymbol{\sigma}$$

- General $\mathcal{J}_{ijkl} = \frac{1+\nu}{E} \left(\frac{1}{2}\delta_{ik}\delta_{jl} + \frac{1}{2}\delta_{il}\delta_{jk}\right) \frac{\nu}{E}\delta_{ij}\delta_{kl}$
- Plane ε

$$\varepsilon_{xx} = \frac{1+\nu}{E} \sigma_{xx} - \frac{\nu}{E} \delta_{xx} \sigma_{kk} = \frac{1+\nu}{E} (\sigma_{xx} - \nu \sigma_{\gamma\gamma})$$

$$\varepsilon_{xy} = \frac{1+\nu}{E} \sigma_{xy} - \frac{\nu}{E} \delta_{xy} \sigma_{kk} = \frac{1+\nu}{E} \sigma_{xy}$$
Greek subscripts for x or y
Roman subscripts for x, y or z

$$\varepsilon_{\alpha\beta} = \frac{1+\nu}{E} (\sigma_{\alpha\beta} - \nu \sigma_{\gamma\gamma} \delta_{\alpha\beta})$$





Static & 2D assumptions: Plane- σ case

 $\Longrightarrow \boldsymbol{\varepsilon}_{\alpha\beta} = \frac{1+\nu}{E} \boldsymbol{\sigma}_{\alpha\beta} - \frac{\nu}{E} \delta_{\alpha\beta} \boldsymbol{\sigma}_{\gamma\gamma}$

– Plane σ :

•
$$\sigma_{xz} = \sigma_{yz} = \sigma_{zz} = 0$$

- Hooke's law
• General
$$\begin{cases} \varepsilon = \mathcal{G} : \sigma \\ \mathcal{G}_{ijkl} = \frac{1+\nu}{E} \left(\frac{1}{2}\delta_{ik}\delta_{jl} + \frac{1}{2}\delta_{il}\delta_{jk}\right) - \frac{\nu}{E}\delta_{ij}\delta_{kl} \\ \bullet \text{ Plane } \sigma \\ - \int_{E} \varepsilon_{xx} = \frac{1+\nu}{E}\sigma_{xx} - \frac{\nu}{E}\delta_{xx}\sigma_{kk} = \frac{\sigma_{xx}}{E} + \sigma_{yy} + \sigma_{zz} = \sigma_{xx} + \sigma_{yy} = \sigma_{aa} \\ \varepsilon_{xy} = \frac{1+\nu}{E}\sigma_{xx} - \frac{\nu}{E}\delta_{xy}\sigma_{kk} = \frac{1+\nu}{E}\sigma_{xx} - \frac{\nu}{E}\sigma_{yy} \end{cases}$$

Greek subscripts for *x* or *y* Roman subscripts for x, y or z

- N.B.:
$$\boldsymbol{\varepsilon}_{\gamma\gamma} = \frac{1+\nu}{E}\boldsymbol{\sigma}_{\gamma\gamma} - \frac{\nu}{E}\boldsymbol{\delta}_{\gamma\gamma}\boldsymbol{\sigma}_{\delta\delta} = \frac{1-\nu}{E}\boldsymbol{\sigma}_{\gamma\gamma}$$

 $2\boldsymbol{\sigma}_{\delta\delta} = 2\boldsymbol{\sigma}_{xx} + 2\boldsymbol{\sigma}_{yy} = 2\boldsymbol{\sigma}_{\gamma\gamma}$





- Static & 2D assumptions: Plane- σ case (2)
 - Plane σ :

•
$$\sigma_{xz} = \sigma_{yz} = \sigma_{zz} = 0$$

- Hooke's law
• General
$$\begin{cases} \varepsilon = \mathcal{G} : \sigma \\ \mathcal{G}_{ijkl} = \frac{1+\nu}{E} \left(\frac{1}{2} \delta_{ik} \delta_{jl} + \frac{1}{2} \delta_{il} \delta_{jk} \right) - \frac{\nu}{E} \delta_{ij} \delta_{kl} \end{cases}$$

• Plane σ

$$-\begin{cases} \varepsilon_{\alpha\beta} = \frac{1+\nu}{E} \sigma_{\alpha\beta} - \frac{\nu}{E} \delta_{\alpha\beta} \sigma_{\gamma\gamma} \\ \varepsilon_{\gamma\gamma} = \frac{1+\nu}{E} \sigma_{\gamma\gamma} - \frac{\nu}{E} \delta_{\gamma\gamma} \sigma_{\delta\delta} = \frac{1-\nu}{E} \sigma_{\gamma\gamma} \\ -\varepsilon_{zz} = \frac{1+\nu}{E} \sigma_{zz}^{0} - \frac{\nu}{E} \sigma_{\gamma\gamma} \\ \Longrightarrow \varepsilon_{zz} = -\frac{\nu}{E} \sigma_{\gamma\gamma} = -\frac{\nu}{1-\nu} \varepsilon_{\gamma\gamma} \end{cases}$$

Greek subscripts for *x* or *y* Roman subscripts for *x*, *y* or *z*





- Static & 2D assumptions: Plane- σ case (3)
 - Plane σ : • $\sigma_{xz} = \sigma_{yz} = \sigma_{zz} = 0$ • $\varepsilon_{zz} = -\frac{\nu}{E}\sigma_{\gamma\gamma} = -\frac{\nu}{1-\nu}\varepsilon_{\gamma\gamma}$ Hooke's law • General $\begin{cases} \boldsymbol{\sigma}_{ij} = \mathcal{H}_{ijkl} \boldsymbol{\varepsilon}_{kl} \\ \mathcal{H}_{ijkl} = \frac{E\nu}{(1+\nu)(1-2\nu)} \delta_{ij} \delta_{kl} + \frac{E}{1+\nu} \left(\frac{1}{2} \delta_{ik} \delta_{jl} + \frac{1}{2} \delta_{il} \delta_{jk}\right) \end{cases}$ Plane σ $\mathbf{\sigma}_{xx} = \frac{E\nu}{(1+\nu)(1-2\nu)} \mathbf{\delta}_{xx} \mathbf{\varepsilon}_{kk} + \frac{E}{1+\nu} \mathbf{\varepsilon}_{xx} \\ \mathbf{\varepsilon}_{kk} = \mathbf{\varepsilon}_{\gamma\gamma} + \mathbf{\varepsilon}_{zz} = \frac{(1-2\nu)}{1-\nu} \mathbf{\varepsilon}_{\gamma\gamma} \\ \mathbf{\sigma}_{xy} = \frac{E\nu}{(1+\nu)(1-2\nu)} \mathbf{\delta}_{xy} \mathbf{\varepsilon}_{kk} + \frac{E}{1+\nu} \mathbf{\varepsilon}_{xy}$ Greek subscripts Greek subscripts for *x* or *y* Roman subscripts for x, y or z $\implies \sigma_{\alpha\beta} = \frac{E\nu}{1-\nu^2}\delta_{\alpha\beta}\varepsilon_{\gamma\gamma} + \frac{E}{1+\nu}\varepsilon_{\alpha\beta}$







- Static & 2D assumptions: Hooke's law
 - Plane σ ($\sigma_{xz} = \sigma_{yz} = \sigma_{zz} = 0$) or plane $\varepsilon (\varepsilon_{xz} = \varepsilon_{yz} = \varepsilon_{zz} = 0 \& \frac{\partial}{\partial z} = 0$)
 - Hooke's law • Plane ε : $\begin{cases}
 \sigma_{\alpha\beta} = \frac{E\nu}{(1+\nu)(1-2\nu)} \delta_{\alpha\beta} \varepsilon_{\gamma\gamma} + \frac{E}{1+\nu} \varepsilon_{\alpha\beta} \\
 \sigma_{zz} = \frac{E\nu}{(1+\nu)(1-2\nu)} \varepsilon_{\gamma\gamma} = \nu \sigma_{\gamma\gamma} \\
 & & & \\ \varepsilon_{\alpha\beta} = \frac{1+\nu}{E} (\sigma_{\alpha\beta} - \nu \sigma_{\gamma\gamma} \delta_{\alpha\beta}) \\
 & & \\ \bullet \text{ Plane } \sigma: & \\ \varepsilon_{zz} = -\frac{\nu}{E} \sigma_{\alpha\beta} - \frac{\nu}{E} \delta_{\alpha\beta} \sigma_{\gamma\gamma} \\
 & \\ \varepsilon_{zz} = -\frac{\nu}{E} \sigma_{\gamma\gamma} = -\frac{\nu}{1-\nu} \varepsilon_{\gamma\gamma} \\
 & \\ & \\ & \\ \bullet \sigma_{\alpha\beta} = \frac{E\nu}{1-\nu^2} \delta_{\alpha\beta} \varepsilon_{\gamma\gamma} + \frac{E}{1+\nu} \varepsilon_{\alpha\beta}
 \end{cases}$
 - Greek subscripts substitute for x or y (Roman for x, y or z)





- Static & 2D assumptions: Linear momentum
 - Plane σ ($\sigma_{xz} = \sigma_{yz} = \sigma_{zz} = 0$) or plane ε ($\varepsilon_{xz} = \varepsilon_{yz} = \varepsilon_{zz} = 0 \& \frac{\partial}{\partial z} = 0$) • Linear momentum $\rho = b + \nabla \cdot \sigma^T$ $0 = b_{\alpha} + \sigma_{\alpha x, x} + \sigma_{\alpha y, y} + \sigma_{\alpha z, z}^{0} \sigma_{\alpha z} = 0 \text{ or } \partial_z = 0$ $\implies \sigma_{\alpha \beta, \beta} + b_{\alpha} = 0$
 - If **b** = 0, there exists an Airy function Φ : $\sigma_{\alpha\beta} = -\Phi_{,\alpha\beta} + \delta_{\alpha\beta}\Phi_{,\gamma\gamma}$

- Indeed
$$\sigma_{\alpha\beta,\beta} = -\Phi_{,\alpha\beta\beta} + \delta_{\alpha\beta}\Phi_{,\gamma\gamma\beta} \Phi_{,\gamma\gamma\alpha} = \Phi_{,\beta\beta\alpha} = \Phi_{,\alpha\beta\beta}$$

 $\implies \sigma_{\alpha\beta,\beta} = -\Phi_{,\alpha\beta\beta} + \Phi_{,\alpha\beta\beta} = 0$

- Airy function:
 - Linear momentum equation is replaced by finding Φ and defining

-
$$\sigma_{\alpha\beta} = -\Phi_{,\alpha\beta} + \delta_{\alpha\beta}\Phi_{,\gamma\gamma}$$





- Static & 2D assumptions: Compatibility
 - Plane σ ($\sigma_{xz} = \sigma_{yz} = \sigma_{zz} = 0$) or plane ε ($\varepsilon_{xz} = \varepsilon_{yz} = \varepsilon_{zz} = 0 \& \frac{\partial}{\partial z} = 0$) • Strain definition

$$\varepsilon_{\alpha\beta} = \frac{1}{2} (\boldsymbol{u}_{\alpha,\beta} + \boldsymbol{u}_{\beta,\alpha})$$

$$\varepsilon_{xx} = \boldsymbol{u}_{x,x}$$

$$\varepsilon_{yy} = \boldsymbol{u}_{y,y}$$

$$\varepsilon_{yy,xx} = \boldsymbol{u}_{y,yxx}$$

$$\varepsilon_{yy,xx} = \boldsymbol{u}_{y,yxx}$$

$$\varepsilon_{xy,xy} = \boldsymbol{u}_{x,yy} + \boldsymbol{u}_{y,xy}$$

$$\implies \varepsilon_{xx,yy} + \varepsilon_{yy,xx} = 2\varepsilon_{xy,xy}$$





Static & 2D assumptions: Bi-harmonic equation in plane- ε ullet

Airy
•
$$\sigma_{\alpha\beta} = -\Phi_{,\alpha\beta} + \delta_{\alpha\beta}\Phi_{,\gamma\gamma} \implies \sigma_{\delta\delta} = -\Phi_{,\delta\delta} + \delta_{\delta\delta}\Phi_{,\gamma\gamma} = \Phi_{,\delta\delta}$$

– Plane
$$\varepsilon$$

•
$$\varepsilon_{\alpha\beta} = \frac{1+\nu}{E} \left(\sigma_{\alpha\beta} - \nu \sigma_{\gamma\gamma} \delta_{\alpha\beta} \right)$$

 $\implies \varepsilon_{\alpha\beta} = \frac{1+\nu}{E} \left[-\Phi_{,\alpha\beta} + (1-\nu) \delta_{\alpha\beta} \Phi_{,\gamma\gamma} \right]$

Compatibility _

$$\boldsymbol{\varepsilon}_{xx,yy} = \frac{1+\nu}{E} \left(-\Phi_{xxyy} + (1-\nu) \left(\Phi_{,xxyy} + \Phi_{,yyyy} \right) \right)$$
$$\boldsymbol{\varepsilon}_{yy,xx} = \frac{1+\nu}{E} \left(-\Phi_{,yyxx} + (1-\nu) \left(\Phi_{,xxxx} + \Phi_{,yyxx} \right) \right)$$
$$\boldsymbol{2\varepsilon}_{xy,xy} = \frac{1+\nu}{E} \left(-2\Phi_{,xyxy} \right)$$

•
$$\boldsymbol{\varepsilon}_{xx,yy} + \boldsymbol{\varepsilon}_{yy,xx} = 2\boldsymbol{\varepsilon}_{xy,xy}$$

$$\implies 0 = \Phi_{,xxyy} + \Phi_{,yyyy} + \Phi_{,xxxx} + \Phi_{,yyxx}$$
$$0 = (\partial_{xx} + \partial_{yy})(\Phi_{,xx} + \Phi_{,yy}) = (\partial_{xx} + \partial_{yy})(\partial_{xx} + \partial_{yy})\Phi$$
$$\implies \nabla^2 \nabla^2 \Phi = 0$$

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Static & 2D assumptions: Bi-harmonic equation in plane- σ

Airy
•
$$\sigma_{\alpha\beta} = -\Phi_{,\alpha\beta} + \delta_{\alpha\beta}\Phi_{,\gamma\gamma} \implies \sigma_{\delta\delta} = -\Phi_{,\delta\delta} + \delta_{\delta\delta}\Phi_{,\gamma\gamma} = \Phi_{,\delta\delta}$$

Plane σ _

•
$$\boldsymbol{\varepsilon}_{\alpha\beta} = \frac{1+\nu}{E} \boldsymbol{\sigma}_{\alpha\beta} - \frac{\nu}{E} \delta_{\alpha\beta} \boldsymbol{\sigma}_{\gamma\gamma}$$

 $\implies \boldsymbol{\varepsilon}_{\alpha\beta} = -\frac{1+\nu}{E} \Phi_{,\alpha\beta} + \frac{1}{E} \delta_{\alpha\beta} \Phi_{,\gamma\gamma}$

Compatibility _

$$\boldsymbol{\varepsilon}_{xx,yy} = \frac{1+\nu}{E} \left(-\Phi_{,xxyy} \right) + \frac{1}{E} \left(\Phi_{,xxyy} + \Phi_{,yyyy} \right)$$
$$\boldsymbol{\varepsilon}_{yy,xx} = \frac{1+\nu}{E} \left(-\Phi_{,yyxx} \right) + \frac{1}{E} \left(\Phi_{,xxxx} + \Phi_{,yyxx} \right)$$
$$\boldsymbol{2}\boldsymbol{\varepsilon}_{xy,xy} = \frac{1+\nu}{E} \left(-2 \Phi_{,xyxy} \right)$$

•
$$\boldsymbol{\varepsilon}_{xx,yy} + \boldsymbol{\varepsilon}_{yy,xx} = 2\boldsymbol{\varepsilon}_{xy,xy}$$

$$\implies 0 = \Phi_{,xxyy} + \Phi_{,yyyy} + \Phi_{,xxxx} + \Phi_{,yyxx}$$
$$0 = (\partial_{xx} + \partial_{yy})(\Phi_{,xx} + \Phi_{,yy}) = (\partial_{xx} + \partial_{yy})(\partial_{xx} + \partial_{yy})\Phi$$
$$\implies \nabla^2 \nabla^2 \Phi = 0$$

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- Static & 2D assumptions: Summary
 - Plane σ ($\sigma_{xz} = \sigma_{yz} = \sigma_{zz} = 0$) or plane $\varepsilon (\varepsilon_{xz} = \varepsilon_{yz} = \varepsilon_{zz} = 0 \& \frac{\partial}{\partial z} = 0$)

Hooke's law
• Plane
$$\varepsilon$$
:

$$\begin{aligned}
\sigma_{\alpha\beta} &= \frac{E\nu}{(1+\nu)(1-2\nu)} \delta_{\alpha\beta} \varepsilon_{\gamma\gamma} + \frac{E}{1+\nu} \varepsilon_{\alpha} \\
\sigma_{zz} &= \frac{E\nu}{(1+\nu)(1-2\nu)} \varepsilon_{\gamma\gamma} = \nu \sigma_{\gamma\gamma} \\
&\& \varepsilon_{\alpha\beta} &= \frac{1+\nu}{E} (\sigma_{\alpha\beta} - \nu \sigma_{\gamma\gamma} \delta_{\alpha\beta}) \\
\bullet & \text{Plane } \sigma : \begin{cases}
\varepsilon_{\alpha\beta} &= \frac{1+\nu}{E} \sigma_{\alpha\beta} - \frac{\nu}{E} \delta_{\alpha\beta} \sigma_{\gamma\gamma} \\
\varepsilon_{zz} &= -\frac{\nu}{E} \sigma_{\gamma\gamma} = -\frac{\nu}{1-\nu} \varepsilon_{\gamma\gamma} \\
&\& \sigma_{\alpha\beta} &= \frac{E\nu}{1-\nu^2} \delta_{\alpha\beta} \varepsilon_{\gamma\gamma} + \frac{E}{1+\nu} \varepsilon_{\alpha\beta}
\end{aligned}$$

- Linear momentum equation
 - Is replaced by finding Φ and defining

$$\boldsymbol{\sigma}_{\alpha\beta} = -\Phi_{,\alpha\beta} + \delta_{\alpha\beta}\Phi_{,\gamma\gamma}$$

 β

- Hooke's law
 - Is replaced by bi-harmonic equations

$$\nabla^2 \nabla^2 \Phi = 0$$





Resolution method



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- Application: infinite plate with a hole (2)
 - For both loading cases: find potential Φ such that
 - Bi-harmonic equation is verified $\ \nabla^2 \nabla^2 \Phi = 0$
 - Stress field $\sigma_{lphaeta}=-\Phi_{,lphaeta}+\delta_{lphaeta}\Phi_{,\gamma\gamma}$ satisfies the boundary conditions
 - Superposition of cases 1 and 2 (see appendix 1)
 - Stresses:

$$\begin{cases} \sigma_{rr} = \frac{\sigma_{\infty}}{2} \left[\left(\frac{4a^2}{r^2} - 1 - \frac{3a^4}{r^4} \right) \cos 2\theta + 1 - \frac{a^2}{r^2} \right] \\ \sigma_{\theta\theta} = \frac{\sigma_{\infty}}{2} \left[\left(1 + \frac{3a^4}{r^4} \right) \cos 2\theta + \frac{a^2}{r^2} + 1 \right] & 3 \\ \sigma_{r\theta} = \frac{\sigma_{\infty}}{2} \left(1 - \frac{3a^4}{r^4} + \frac{2a^2}{r^2} \right) \sin 2\theta & 2.5 \\ \bullet & \text{For } \theta = 0; \ \sigma_{yy} = \sigma_{\theta\theta} & 1.5 \\ \implies \sigma_{yy} (x, y = 0) = \frac{\sigma_{\infty}}{2} \left(2 + \frac{3a^4}{x^4} + \frac{a^2}{x^2} \right) \begin{array}{c} 1 \\ 0.5 \\ \bullet \end{array} \right]$$

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- Application: infinite plate with an elliptical hole
 - Elliptical coordinates

 $\begin{aligned} x &= c \cosh \alpha \cos \beta \\ y &= c \sinh \alpha \sin \beta \end{aligned}$

with, on the boundary,

$$\int c \cosh \alpha_0 = a$$
$$\int c \sinh \alpha_0 = b$$



Resolution using complex functions of z (see next slides) _

 $z = c \cosh \zeta$ with the complex variable $\zeta = \alpha + i \beta$

• 1913, Inglis
$$\sigma_{\max} = \sigma_{yy}(a, 0) = \sigma_{\infty} \left(1 + \frac{2a}{b}\right)$$

Stress concentration factor

•
$$K_{\text{ell}} = 1 + \frac{2a}{b} = 1 + 2\sqrt{\frac{a}{\rho}}$$

with ρ the curvature radius at the tip

Singularity when $\rho \rightarrow 0$







Fracture modes

- 1957, Irwin, 3 fracture modes



Resolution method

- Application: cracked body
 - Change of variables
 - Complex variables $\zeta = x + iy = r \exp i\theta$
 - Complex functions $z\left(\zeta\right) = \alpha\left(x, \, y\right) + i\beta\left(x, \, y\right)$ with α & β real functions

 $y \uparrow \qquad y \uparrow \qquad y \uparrow \qquad y \uparrow \qquad y \uparrow \qquad \zeta \downarrow \qquad \zeta \downarrow$

Interest





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- Method for linear and elastic materials ۲
 - Use of
 - Complex variables

$$\zeta = x + iy = r \exp i\theta$$

1

Complex functions •

(~)

$$\begin{split} z\left(\zeta\right) &= \alpha\left(x,\,y\right) + i\beta\left(x,\,y\right) \\ \text{with } \alpha \And \beta \text{ real functions} \begin{cases} \alpha &= \mathcal{R}\left(z\left(\zeta\right)\right) \\ \beta &= \mathcal{I}\left(z\left(\zeta\right)\right) \end{cases} \end{split}$$



If z is differentiable (analytic) $\implies z'(\zeta) = \partial_x z = -i\partial_y z$ —

)

This yields the Cauchy-Riemann relations •

$$\partial_x z = \alpha_{,x} + i\beta_{,x} = -i\partial_y z = -i\alpha_{,y} + \beta_{,y} \Longrightarrow \alpha_{,x} = \beta_{,y} \& \alpha_{,y} = -\beta_{,x}$$

And the functions satisfy the Laplace equation •

$$\alpha_{,xx} = \beta_{,xy} = -\alpha_{,yy} \& \beta_{,xx} = -\alpha_{,xy} = -\beta_{,yy} \implies \nabla^2 \alpha = \nabla^2 \beta = 0$$







- Method for linear and elastic materials (2)
 - What becomes of the bi-harmonic equation?
 - Change of variables: $x, y \rightarrow \zeta, \overline{\zeta}$

- The Airy function $\Phi(x, y) = \Phi(\zeta, \overline{\zeta})$
 - Let us define Ψ such that: $\nabla^2 \Phi\left(\zeta,\,\bar{\zeta}
 ight) = \Psi\left(\zeta,\,\bar{\zeta}
 ight)$
 - The bi-harmonic equation is rewritten $\nabla^2 \nabla^2 \Phi\left(\zeta, \, \bar{\zeta}\right) = \nabla^2 \Psi\left(\zeta, \, \bar{\zeta}\right) = 0$
- Since Ψ satisfies the Laplace equation, it is the real part of a function $z(\zeta)$:
 - $-\Psi(\zeta, \bar{\zeta}) = \mathcal{R}z = \frac{z + \bar{z}}{2} \quad \text{with } \bar{z} = z(\bar{\zeta}) \text{ a function of } \bar{\zeta} \text{ only}$

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$$\begin{cases} \Phi_{,xx} = \partial_x \left(\Phi_{,\zeta} + \Phi_{,\bar{\zeta}} \right) = \Phi_{,\zeta\zeta} + 2\Phi_{,\zeta\bar{\zeta}} + \Phi_{,\bar{\zeta}\bar{\zeta}} \\ \Phi_{,yy} = \partial_y \left(i\Phi_{,\zeta} - i\Phi_{,\bar{\zeta}} \right) = -\Phi_{,\zeta\zeta} + 2\Phi_{,\zeta\bar{\zeta}} - \Phi_{,\bar{\zeta}\bar{\zeta}} \end{cases}$$

$$\implies \nabla^2 \Phi = \Phi_{,xx} + \Phi_{,yy} = 4\Phi_{,\zeta\bar{\zeta}} = \Psi$$







- Method for linear and elastic materials (3)
 - What becomes of the bi-harmonic equation?
 - Change of variables: $x, y \rightarrow \zeta, \overline{\zeta}$

$$\begin{cases} \zeta = x + iy \\ \bar{\zeta} = x - iy \end{cases} \implies \begin{cases} \partial_x = \partial_{\zeta}\zeta_{,x} + \partial_{\bar{\zeta}}\bar{\zeta}_{,x} = \partial_{\zeta} + \partial_{\bar{\zeta}} \\ \partial_y = \partial_{\zeta}\zeta_{,y} + \partial_{\bar{\zeta}}\bar{\zeta}_{,y} = i\partial_{\zeta} - i\partial_{\bar{\zeta}} \end{cases}$$

• The Airy function $\Phi(x, y) = \Phi(\zeta, \overline{\zeta})$ with $\nabla^2 \Phi(\zeta, \overline{\zeta}) = \Psi(\zeta, \overline{\zeta})$

$$\implies \left\{ \begin{array}{l} \Psi\left(\zeta,\,\bar{\zeta}\right) = \mathcal{R}z = \frac{z+\bar{z}}{2} \\ \nabla^2 \Phi = \Phi_{,xx} + \Phi_{,yy} = 4\Phi_{,\zeta\bar{\zeta}} = \Psi \end{array} \right.$$

• Let $4\Omega'(\zeta) = z(\zeta)$ and $\omega(\zeta)$ be the unknown functions

$$\implies 4\Phi_{,\zeta\bar{\zeta}} = \frac{z+\bar{z}}{2} = 2(\Omega' + \bar{\Omega}')$$

$$4\Phi_{,\zeta} = 2(\Omega'\bar{\zeta} + \bar{\Omega}) + f(\zeta)$$

$$4\Phi = 2(\Omega\bar{\zeta} + \bar{\Omega}\zeta) + \int f(\zeta) + 2\omega(\bar{\zeta})$$

$$\implies \Phi = \frac{(\Omega\bar{\zeta} + \bar{\Omega}\zeta) + \omega + \bar{\omega}}{2}$$



• $\Omega(\zeta)$ and $\omega(\zeta)$ are the new and only unknown functions



- Method for linear and elastic materials (4)
 - What become of the stresses?

• Since
$$\Phi_{,\zeta\zeta} = \frac{\bar{\zeta}\Omega'' + \omega''}{2}$$
, $\Phi_{,\zeta\bar{\zeta}} = \frac{\Omega' + \bar{\Omega}'}{2}$ & $\Phi_{,\bar{\zeta}\bar{\zeta}} = \frac{\zeta\bar{\Omega}'' + \bar{\omega}''}{2}$

The stresses are rewritten •

$$\begin{cases} \boldsymbol{\sigma}_{xx} = \Phi_{,yy} = -\Phi_{,\zeta\zeta} + 2\Phi_{,\zeta\bar{\zeta}} - \Phi_{,\bar{\zeta}\bar{\zeta}} = \Omega' + \bar{\Omega}' - \frac{\bar{\zeta}\Omega'' + \omega'' + \zeta\bar{\Omega}'' + \bar{\omega}''}{2} \\ \boldsymbol{\sigma}_{yy} = \Phi_{,xx} = \Phi_{,\zeta\zeta} + 2\Phi_{,\zeta\bar{\zeta}} + \Phi_{,\bar{\zeta}\bar{\zeta}} = \Omega' + \bar{\Omega}' + \frac{\bar{\zeta}\Omega'' + \omega'' + \zeta\bar{\Omega}'' + \bar{\omega}''}{2} \\ \boldsymbol{\sigma}_{xy} = -\Phi_{,xy} = -\partial_y \left(\Phi_{,\zeta} + \Phi_{,\bar{\zeta}} \right) = -i\Phi_{,\zeta\zeta} + i\Phi_{,\bar{\zeta}\bar{\zeta}} = i\frac{\zeta\bar{\Omega}'' + \bar{\omega}'' - \bar{\zeta}\Omega'' - \omega''}{2} \end{cases}$$







- Method for linear and elastic materials (5)
 - Displacements can be deduced from the stresses

• Complex form:
$$\boldsymbol{u} = \boldsymbol{u}_x \left(x, \, y \right) + i \boldsymbol{u}_y \left(x, \, y \right) = \boldsymbol{u} \left(\zeta, \, \bar{\zeta} \right)$$

• Using Hooke's law (see appendix 2)

$$\begin{aligned} &- u = -\frac{1+\nu}{E} \left(\zeta \bar{\Omega}' + \bar{\omega}' - \kappa \Omega \left(\zeta \right) \right) \\ &- \text{Plane } \sigma \colon \kappa = \frac{3-\nu}{1+\nu} \end{aligned}$$

- Plane
$$\varepsilon$$
: $\kappa = 3 - 4\nu$







• Static & 2D assumptions: Summary

- Plane σ ($\sigma_{xz} = \sigma_{yz} = \sigma_{zz} = 0$) or plane $\varepsilon (\varepsilon_{xz} = \varepsilon_{yz} = \varepsilon_{zz} = 0 \& \frac{\partial}{\partial z} = 0$)

- Hooke's law & linear momentum equations replaced by

$$\begin{cases} \boldsymbol{\sigma}_{\alpha\beta} = -\Phi_{,\alpha\beta} + \delta_{\alpha\beta}\Phi_{,\gamma\gamma} \\ \nabla^2 \nabla^2 \Phi = 0 \end{cases}$$

- One unknown function: finding Φ and defining so that BCs are satisfied
- Complex variable

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• No more differential equation if we use

$$\Phi = \frac{(\Omega \bar{\zeta} + \bar{\Omega} \zeta) + \omega + \bar{\omega}}{2}$$



• $\Omega(\zeta)$ and $\omega(\zeta)$ are the unknown functions which should satisfy the BCs

$$\begin{cases} \boldsymbol{\sigma}_{xx} = \ \Omega' + \bar{\Omega}' - \frac{\bar{\zeta}\Omega'' + \omega'' + \zeta\bar{\Omega}'' + \bar{\omega}''}{2} \\ \boldsymbol{\sigma}_{yy} = \ \Omega' + \bar{\Omega}' + \frac{\bar{\zeta}\Omega'' + \omega'' + \zeta\bar{\Omega}'' + \bar{\omega}''}{2} \\ \boldsymbol{\sigma}_{xy} = \ i\frac{\zeta\bar{\Omega}'' + \bar{\omega}'' - \bar{\zeta}\Omega'' - \omega''}{2} \end{cases} \quad \begin{cases} \boldsymbol{u} = -\frac{1+\nu}{E} \left(\zeta\bar{\Omega}' + \bar{\omega}' - \kappa\Omega\left(\zeta\right)\right) \\ \kappa = \frac{3-\nu}{1+\nu} \\ \kappa = 3-4\nu \end{cases} \text{Plane } \boldsymbol{\varepsilon}. \end{cases}$$





Asymptotic solution

y

x

- Application: cracked body
 - Choice of complex functions
 - Owing to the discontinuity for $\theta = \pi$ we choose

$$\left\{ \begin{array}{l} \Omega(\zeta) = \sum_{\lambda} \left(C_{1}^{(\lambda)} + iC_{2}^{(\lambda)} \right) \zeta^{\lambda+1} = \sum_{\lambda} \left(C_{1}^{(\lambda)} + iC_{2}^{(\lambda)} \right) r^{\lambda+1} \exp\left(i\theta(\lambda+1)\right) \\ \omega'(\zeta) = \sum_{\lambda} \left(C_{3}^{(\lambda)} + iC_{4}^{(\lambda)} \right) \zeta^{\lambda+1} = \sum_{\lambda} \left(C_{3}^{(\lambda)} + iC_{4}^{(\lambda)} \right) r^{\lambda+1} \exp\left(i\theta(\lambda+1)\right) \\ \end{array} \right.$$

Indeed, for λ =-1/2 the functions are discontinuous

- Conditions on λ
 - Stress σ in $\Omega' \implies$ displacement u in $\Omega \implies u \propto r^{\lambda+1}$

Displacement *u* remains finite $\implies \lambda > -1$

Crack is stress free

$$\begin{cases} \boldsymbol{\sigma}_{yy} \left(\boldsymbol{\theta} = \pm \pi \right) = 0 \\ \boldsymbol{\sigma}_{xy} \left(\boldsymbol{\theta} = \pm \pi \right) = 0 \end{cases} \xrightarrow{\boldsymbol{y}}_{\boldsymbol{x}} \boldsymbol{\theta}$$





- Application to mode I (opening)
 - New constraints

$$\left\{ \begin{array}{l} \Omega(\zeta) = \sum_{\lambda > -1} \left(C_1^{(\lambda)} + iC_2^{(\lambda)} \right) \zeta^{\lambda+1} = \sum_{\lambda > -1} \left(C_1^{(\lambda)} + iC_2^{(\lambda)} \right) r^{\lambda+1} \exp\left(i\theta(\lambda+1)\right) \\ \omega'(\zeta) = \sum_{\lambda > -1} \left(C_3^{(\lambda)} + iC_4^{(\lambda)} \right) \zeta^{\lambda+1} = \sum_{\lambda > -1} \left(C_3^{(\lambda)} + iC_4^{(\lambda)} \right) r^{\lambda+1} \exp\left(i\theta(\lambda+1)\right) \\ \end{array} \right.$$

• Symmetry:

$$\begin{array}{c|c}
\mathbf{y} & \mathbf{y} \\
\mathbf{y} & \mathbf{y} \\
\mathbf{y} & \mathbf{y} \\
\mathbf{y} \\
\mathbf{y} \\
\mathbf{x} \\
\mathbf{x} \\
\mathbf{u}_{x} (\theta > 0) = \mathbf{u}_{x} (\theta < 0) \\
\mathbf{u}_{y} (\theta > 0) = -\mathbf{u}_{y} (\theta < 0) \\
\mathbf{u}_{y} (\theta > 0) = -\mathbf{u}_{y} (\theta < 0) \\
\mathbf{u}_{y} (\theta < 0) = -\mathbf{u}_{y} (\theta < 0)
\end{array}$$

 $C_2^{(\lambda)} = C_4^{(\lambda)} = 0$

 $\Re \Omega(\zeta), \Re \overline{\omega}' \text{ in } \cos(\theta) \text{ and } \Im \Omega(\zeta), \Im \overline{\omega}' \text{ in } \sin(\theta)$





- Application to mode I (opening) (2)
 - New constraints (2)

$$\left\{ \begin{array}{l} \Omega(\zeta) = \sum_{\lambda \succ -1} C_1^{(\lambda)} \, \zeta^{\lambda+1} = \sum_{\lambda \succ -1} C_1^{(\lambda)} \, r^{\lambda+1} \exp\bigl(i\theta(\lambda+1)\bigr) \\ \omega'(\zeta) = \sum_{\lambda \succ -1} C_3^{(\lambda)} \, \zeta^{\lambda+1} = \sum_{\lambda \succ -1} C_3^{(\lambda)} \, r^{\lambda+1} \exp\bigl(i\theta(\lambda+1)\bigr) \end{array} \right.$$



• Stress field:

$$\begin{cases} \boldsymbol{\sigma}_{xx} = \Omega' + \bar{\Omega}' - \frac{\bar{\zeta}\Omega'' + \omega'' + \zeta\bar{\Omega}'' + \bar{\omega}''}{2} \\ \boldsymbol{\sigma}_{yy} = \Omega' + \bar{\Omega}' + \frac{\bar{\zeta}\Omega'' + \omega'' + \zeta\bar{\Omega}'' + \bar{\omega}''}{2} \\ \boldsymbol{\sigma}_{xy} = i\frac{\zeta\bar{\Omega}'' + \bar{\omega}'' - \bar{\zeta}\Omega'' - \omega''}{2} \end{cases}$$

$$\sigma_{xx} = \sum_{\lambda > -1} (\lambda + 1) \left[C_1^{(\lambda)} (\zeta^{\lambda} + \bar{\zeta}^{\lambda}) - \frac{C_1^{(\lambda)} \lambda (\bar{\zeta} \zeta^{\lambda - 1} + \zeta \bar{\zeta}^{\lambda - 1}) + C_3^{(\lambda)} (\zeta^{\lambda} + \bar{\zeta}^{\lambda})}{2} \right]$$

$$\sigma_{yy} = \sum_{\lambda > -1} (\lambda + 1) \left[C_1^{(\lambda)} (\zeta^{\lambda} + \bar{\zeta}^{\lambda}) + \frac{C_1^{(\lambda)} \lambda (\bar{\zeta} \zeta^{\lambda - 1} + \zeta \bar{\zeta}^{\lambda - 1}) + C_3^{(\lambda)} (\zeta^{\lambda} + \bar{\zeta}^{\lambda})}{2} \right]$$

$$\sigma_{xy} = \sum_{\lambda > -1} (\lambda + 1) \left[-i \frac{C_1^{(\lambda)} \lambda (\bar{\zeta} \zeta^{\lambda - 1} - \zeta \bar{\zeta}^{\lambda - 1}) + C_3^{(\lambda)} (\zeta^{\lambda} - \bar{\zeta}^{\lambda})}{2} \right]$$



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Asymptotic solution

- Application to mode I (opening) (3)
 - New constraints (3)
 - Stress field:

Stress field:

$$\begin{cases}
\boldsymbol{\sigma}_{xx} = \sum_{\lambda \geq -1} (\lambda + 1) \left[C_{1}^{(\lambda)} (\zeta^{\lambda} + \bar{\zeta}^{\lambda}) - \frac{C_{1}^{(\lambda)} \lambda (\bar{\zeta} \zeta^{\lambda - 1} + \zeta \bar{\zeta}^{\lambda - 1}) + C_{3}^{(\lambda)} (\zeta^{\lambda} + \bar{\zeta}^{\lambda})}{2} \right] \\
\boldsymbol{\sigma}_{yy} = \sum_{\lambda \geq -1} (\lambda + 1) \left[C_{1}^{(\lambda)} (\zeta^{\lambda} + \bar{\zeta}^{\lambda}) + \frac{C_{1}^{(\lambda)} \lambda (\bar{\zeta} \zeta^{\lambda - 1} + \zeta \bar{\zeta}^{\lambda - 1}) + C_{3}^{(\lambda)} (\zeta^{\lambda} + \bar{\zeta}^{\lambda})}{2} \right] \\
\boldsymbol{\sigma}_{xy} = \sum_{\lambda \geq -1} (\lambda + 1) \left[-i \frac{C_{1}^{(\lambda)} \lambda (\bar{\zeta} \zeta^{\lambda - 1} - \zeta \bar{\zeta}^{\lambda - 1}) + C_{3}^{(\lambda)} (\zeta^{\lambda} - \bar{\zeta}^{\lambda})}{2} \right]$$

 $\zeta = r \exp(i\theta)$ $\bar{\zeta} = r \exp(-i\theta)$

$$\sigma_{xx} = \sum_{\lambda > -1} (\lambda + 1) r^{\lambda} \left[C_{1}^{(\lambda)} 2\cos(\lambda\theta) - \frac{2C_{1}^{(\lambda)}\lambda\cos((\lambda - 2)\theta) + 2C_{3}^{(\lambda)}\cos(\lambda\theta)}{2} \right]$$
$$\sigma_{yy} = \sum_{\lambda > -1} (\lambda + 1) r^{\lambda} \left[C_{1}^{(\lambda)} 2\cos(\lambda\theta) + \frac{2C_{1}^{(\lambda)}\lambda\cos((\lambda - 2)\theta) + 2C_{3}^{(\lambda)}\cos(\lambda\theta)}{2} \right]$$
$$\sigma_{xy} = \sum_{\lambda > -1} (\lambda + 1) r^{\lambda} \left[-i\frac{2C_{1}^{(\lambda)}\lambda i\sin((\lambda - 2)\theta) + 2C_{3}^{(\lambda)}i\sin(\lambda\theta)}{2} \right]$$





• Application to mode I (opening) (4)

• Stress free crack: $\begin{cases} \sigma_{yy} (\theta = \pm \pi) = 0 & \text{ for } \theta \\ \sigma_{xy} (\theta = \pm \pi) = 0 & \text{ for } \theta \\ \hline & & & & & & & \\ \end{array}$

$$2C_1^{(\lambda)}\cos(\pm\lambda\pi) + C_1^{(\lambda)}\lambda\cos(\pm\lambda\pi) + C_3^{(\lambda)}\cos(\pm\lambda\pi) = 0$$
$$C_1^{(\lambda)}\lambda\sin(\pm\lambda\pi) + C_3^{(\lambda)}\sin(\pm\lambda\pi) = 0$$

$$\begin{pmatrix} (2+\lambda)\cos(\lambda\pi) & \cos(\lambda\pi) \\ \lambda\sin(\lambda\pi) & \sin(\lambda\pi) \end{pmatrix} \begin{pmatrix} C_1^{(\lambda)} \\ C_3^{(\lambda)} \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \end{pmatrix}$$





- Application to mode I (opening) (5)
 - Solution
 - System of equations

$$\begin{pmatrix} (2+\lambda)\cos(\lambda\pi) & \cos(\lambda\pi) \\ \lambda\sin(\lambda\pi) & \sin(\lambda\pi) \end{pmatrix} \begin{pmatrix} C_1^{(\lambda)} \\ C_3^{(\lambda)} \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \end{pmatrix}$$

Non trivial solution

$$0 = \begin{vmatrix} (2+\lambda)\cos(\lambda\pi) & \cos(\lambda\pi) \\ \lambda\sin(\lambda\pi) & \sin(\lambda\pi) \end{vmatrix} = \sin(2\lambda\pi)$$

$$\lambda = n/2$$
, with $n = -1, 0, 1, \dots$ since $\lambda > -1$

• Displacements in $r^{\lambda+1}$ and stresses in r^{λ}

 \implies the dominant term **near the crack tip** is obtained for $\lambda = -1/2$

estimate only this term is considered in the **asymptotic** solution

• Determination of the constants:

$$\sigma_{xy} \left(\theta \pm \pi\right) = 0 \implies C_1^{(\lambda)} \lambda \sin(\pm \lambda \pi) + C_3^{(\lambda)} \sin(\pm \lambda \pi) = 0 \text{ for } \lambda = \frac{n}{2}$$
$$\longrightarrow -\lambda C_1^{(\lambda)} = C_3^{(\lambda)} \implies C_1^{\left(-\frac{1}{2}\right)} = 2C_3^{\left(-\frac{1}{2}\right)}$$







- Application to mode I (opening) (6)
 - Stress field:

$$\begin{split} & \left\{ \begin{array}{l} \sigma_{xx} = \sum_{\lambda > -1} (\lambda + 1)r^{\lambda} \left[2C_{1}^{(\lambda)} \cos(\lambda\theta) - C_{1}^{(\lambda)} \lambda \cos((\lambda - 2)\theta) - C_{3}^{(\lambda)} \cos(\lambda\theta) \right] \\ \sigma_{yy} = \sum_{\lambda > -1} (\lambda + 1)r^{\lambda} \left[2C_{1}^{(\lambda)} \cos(\lambda\theta) + C_{1}^{(\lambda)} \lambda \cos((\lambda - 2)\theta) + C_{3}^{(\lambda)} \cos(\lambda\theta) \right] \\ \sigma_{xy} = \sum_{\lambda > -1} (\lambda + 1)r^{\lambda} \left[C_{1}^{(\lambda)} \lambda \sin((\lambda - 2)\theta) + C_{3}^{(\lambda)} \sin(\lambda\theta) \right] \\ & \left\{ \begin{array}{l} \sigma_{xy} = \sum_{\lambda > -1} (\lambda + 1)r^{\lambda} \left[C_{1}^{(\lambda)} \lambda \sin((\lambda - 2)\theta) + C_{3}^{(\lambda)} \sin(\lambda\theta) \right] \\ \sigma_{xy} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots \end{array} \right. \\ & \left\{ \begin{array}{l} \sigma_{xx} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots \end{array} \right. \\ \left\{ \begin{array}{l} \sigma_{xx} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots \end{array} \right. \\ & \left\{ \begin{array}{l} \sigma_{xy} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots \end{array} \right. \\ \left\{ \begin{array}{l} \sigma_{xy} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots \end{array} \right. \\ & \left\{ \begin{array}{l} \alpha_{xy} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots \end{array} \right. \\ & \left\{ \begin{array}{l} \alpha_{xy} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots \end{array} \right. \\ & \left\{ \begin{array}{l} \alpha_{xy} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots \end{array} \right. \\ & \left\{ \begin{array}{l} \alpha_{xy} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots \end{array} \right. \\ & \left\{ \left\{ \lambda + 1 \right\} C_{1}^{(\lambda)} r^{\lambda} \left[2\cos(\lambda\theta) + \lambda \left(\cos((\lambda - 2)\theta) - \cos(\lambda\theta) \right) \right] \right\} \\ & \left\{ \begin{array}{l} \sigma_{xy} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots \end{array} \right. \\ & \left\{ \lambda + 1 \right\} r^{\lambda} C_{1}^{(\lambda)} \lambda \left[\sin((\lambda - 2)\theta) - \sin(\lambda\theta) \right] \\ & \left\{ \begin{array}{l} \sin a - \sin b = 2\sin \frac{a - b}{2} \cos \frac{a + b}{2} \end{array} \right\} \\ & \left\{ \left\{ n - \frac{a - b}{2} \cos \frac{a + b}{2} \right\} \right\} \\ & \left\{ n - \frac{a - b}{2} \cos \frac{a + b}{2} \right\} \\ & \left\{ n - \frac{a - b}{2} \cos \frac{a - b}{2} \cos \frac{a + b}{2} \right\} \\ & \left\{ n - \frac{a - b}{2} \cos \frac{a - b}{$$





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- Application to mode I (opening) (7)
 - Stress field summary:
 - Symmetry:



• Leads to $C_2^{(\lambda)} = C_4^{(\lambda)} = 0$ & relation between $C_1^{(\lambda)} \& -\lambda C_1^{(\lambda)} = C_3^{(\lambda)}$

$$\left\{ \begin{array}{l} \Omega(\zeta) = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} C_{1}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} C_{1}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \\ \omega'(\zeta) = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} -\lambda C_{1}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} -\lambda C_{1}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \\ \\ \left. \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} \sigma_{xx} = \int_{\sqrt{r}} \cos \frac{\theta}{2} \left[1 - \sin \frac{\theta}{2} \sin \frac{3\theta}{2} \right] + A(\theta) r^{0} + B(\theta) \sqrt{r} \dots \\ \\ \sigma_{yy} = \int_{\sqrt{r}} \cos \frac{\theta}{2} \left[1 + \sin \frac{\theta}{2} \sin \frac{3\theta}{2} \right] + C(\theta) r^{0} + D(\theta) \sqrt{r} \dots \\ \\ \sigma_{xy} = \int_{\sqrt{r}} \cos \frac{\theta}{2} \sin \frac{\theta}{2} \cos \frac{3\theta}{2} + E(\theta) r^{0} + F(\theta) \sqrt{r} \dots \end{array} \right.$$







- Application to mode I (opening) (8)
 - Asymptotic stress field

•
$$\sigma_{yy} = \frac{C_1}{\sqrt{r}} \cos \frac{\theta}{2} \left[1 + \sin \frac{\theta}{2} \sin \frac{3\theta}{2} \right] + C(\theta)r^0 + D(\theta)\sqrt{r} \dots$$



- For mode I, the stress concentration is obtained for $\sigma_{yy}(\theta = 0)$ •
- As the stress tends to infinity, the Stress Intensity Factor (SIF) is defined as •

$$\implies K_I = \lim_{r \to 0} \left(\sqrt{2\pi r} \, \boldsymbol{\sigma}_{yy}^{\text{mode I}} \big|_{\theta=0} \right) = C_1 \sqrt{2\pi}$$

- $C_1 \& \text{so } K_1 \text{ depend on both the geometry and loading condition}$
- Unit of the SIFs in Pa m^{1/2} •







- Application to mode I (opening) (9)
 - Asymptotic stress field (dominant terms)











Fracture Mechanics - LEFM - Asymptotic Solution



- Application to mode I (opening) (10)
 - Displacement field:

•
$$\begin{cases} \Omega(\zeta) = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} C_{1}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} C_{1}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \\ \omega'(\zeta) = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} -\lambda C_{1}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} -\lambda C_{1}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \end{cases}$$

•
$$\boldsymbol{u} = -\frac{1+\nu}{E} \left(\zeta \bar{\Omega}' + \bar{\omega}' - \kappa \Omega \left(\zeta \right) \right) \begin{cases} \kappa = \frac{3-\nu}{1+\nu} & \text{Plane } \sigma: \\ \kappa = 3 - 4\nu & \text{Plane } \varepsilon. \end{cases}$$

$$u_{x} + iu_{y}$$

$$= \frac{1+\nu}{E} \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} C_{1}^{(\lambda)} r^{\lambda+1} [\kappa \exp(i(\lambda+1)\theta) - (\lambda+1)\exp(i(1-\lambda)\theta) + \lambda \exp(-i(\lambda+1)\theta)]$$

$$\& C_{1}^{\left(-\frac{1}{2}\right)} = \frac{K_{I}}{\sqrt{2\pi}}$$





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- Application to mode I (opening) (11)
 - Asymptotic displacements (dominant terms)

$$\begin{bmatrix} \mathbf{u}_x = K_I \frac{1+\nu}{E} \sqrt{\frac{r}{2\pi}} \cos \frac{\theta}{2} \left[\kappa - 1 + 2 \sin^2 \frac{\theta}{2} \right] \\ \mathbf{u}_y = K_I \frac{1+\nu}{E} \sqrt{\frac{r}{2\pi}} \sin \frac{\theta}{2} \left[\kappa + 1 - 2 \cos^2 \frac{\theta}{2} \right] \end{bmatrix}$$

• With for plane $\sigma:\ \kappa=\frac{3-\nu}{1+\nu}$ & for plane $\epsilon:\kappa=3-4\nu$





Fracture Mechanics - LEFM - Asymptotic Solution

- Application to mode I (opening) (12)
 - Validity of the asymptotic solution
 - This solution is only valid in
 - a restricted region









Asymptotic solution

- Application to mode II (sliding)
 - New constraints:
 - Symmetry:



 $\begin{cases} \boldsymbol{\sigma}_{zz} = 0 \text{ or } \boldsymbol{\varepsilon}_{zz} = 0 \\ \boldsymbol{\sigma}_{yy} \left(\boldsymbol{\theta} = \pm \pi \right) = 0 \\ \boldsymbol{\sigma}_{xy} \left(\boldsymbol{\theta} = \pm \pi \right) = 0 \\ \boldsymbol{u}_{x} \left(\boldsymbol{\theta} > 0 \right) = -\boldsymbol{u}_{x} \left(\boldsymbol{\theta} < 0 \right) \\ \boldsymbol{u}_{y} \left(\boldsymbol{\theta} > 0 \right) = \boldsymbol{u}_{y} \left(\boldsymbol{\theta} < 0 \right) \end{cases}$

• Leads to $C_1^{(\lambda)} = C_3^{(\lambda)} = 0$ & relation between $C_2^{(\lambda)} \& C_4^{(\lambda)}$

$$\begin{aligned}
\Omega(\zeta) &= \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{2}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{2}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \\
\omega'(\zeta) &= \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \zeta^{\lambda+1} = \sum_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} r^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \tau^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \tau^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \tau^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} e^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} e^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} e^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} e^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} e^{\lambda+1} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \exp(i\theta(\lambda+1)) \\
&= \int_{\lambda = -\frac{1}{2}, 0, \frac{1}{2}, \dots} iC_{4}^{(\lambda)} \exp(i\theta(\lambda+$$



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• Application to mode II (sliding) (2)



- Application to mode II (sliding) (3)
 - Asymptotic displacements (dominant terms)

$$\begin{bmatrix} \boldsymbol{u}_x = K_{II} \frac{1+\nu}{E} \sqrt{\frac{r}{2\pi}} \sin \frac{\theta}{2} \left[\kappa + 1 + 2 \cos^2 \frac{\theta}{2} \right] \\ \boldsymbol{u}_y = K_{II} \frac{1+\nu}{E} \sqrt{\frac{r}{2\pi}} \cos \frac{\theta}{2} \left[1 - \kappa + 2 \sin^2 \frac{\theta}{2} \right] \end{bmatrix}$$

• With for plane σ : $\kappa = \frac{3-\nu}{1+\nu}$ & for plane ϵ : $\kappa = 3-4\nu$



- Application to mode III (shearing)
 - Mode III: $\begin{array}{c|c} \mathbf{y} \\ \mathbf{0} \\ \mathbf{y} \\ \mathbf{0} \\ \mathbf{x} \end{array} \qquad \left\{ \begin{array}{c} \boldsymbol{\sigma}_{xx} = \boldsymbol{\sigma}_{xy} = \boldsymbol{\sigma}_{yz} = \mathbf{0} \\ \boldsymbol{u}_{y} = \boldsymbol{u}_{x} = \mathbf{0} \\ \boldsymbol{u}_{z} \ (\theta > 0) = -\boldsymbol{u}_{z} \ (\theta < 0) \end{array} \right.$
 - Equations
 - Hooke's law

Equilibrium

$$\boldsymbol{\varepsilon}_{\alpha z} = \frac{1}{2} \boldsymbol{u}_{z,\alpha} \implies \boldsymbol{\sigma}_{\alpha z} = \frac{E}{1+\nu} \boldsymbol{\varepsilon}_{\alpha z} = \frac{E}{2(1+\nu)} \boldsymbol{u}_{z,\alpha}$$
$$\boldsymbol{\sigma}_{\alpha z,\alpha} = 0 \implies \nabla^2 \boldsymbol{u}_z = 0$$

- Laplace equation satisfied $\implies u_z$ is the imaginary part of a function $z(\zeta)$

• Choice of function $z = \sum_{\lambda} C^{(\lambda)} \zeta^{\lambda} = \sum_{\lambda} C^{(\lambda)} r^{\lambda} \exp(i\lambda\theta)$

•
$$\boldsymbol{u}_{z} \left(\theta > 0 \right) = - \boldsymbol{u}_{z} \left(\theta < 0 \right)$$

 $\implies u_z$ is the imaginary part of z, and $C^{(\lambda)}$ is real

$$\implies \boldsymbol{u}_z = \sum_{\lambda} C^{(\lambda)} r^{\lambda} \sin(\lambda \theta)$$





- Application to mode III (shearing) (2)
 - Stress field:

•
$$\int_{C} \boldsymbol{u}_{z} = \sum_{\lambda} C^{(\lambda)} r^{\lambda} \sin(\lambda \theta)$$
•
$$\int_{C} \boldsymbol{\sigma}_{\alpha z} = \frac{E}{1+\nu} \boldsymbol{\varepsilon}_{\alpha z} = \frac{E}{2(1+\nu)} \boldsymbol{u}_{z,\alpha}$$



Cylindrical CV



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• Application to mode III (shearing) (3)

- Mode III:

$$\mathbf{y}$$

 \mathbf{r}
 θ
 \mathbf{x}
 \mathbf{r}
 $\mathbf{\theta}$
 \mathbf{x}
 \mathbf{r}
 $\mathbf{\theta}$
 \mathbf{x}
 \mathbf{r}
 $\mathbf{\theta}$
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 $\mathbf{\theta}$
 \mathbf{x}
 \mathbf{r}
 \mathbf{r}
 \mathbf{r}
 \mathbf{r}
 $\mathbf{\theta}$
 \mathbf{x}
 \mathbf{r}
 $\mathbf{$

- Resolution (Hooke's law & equilibrium)

•
$$\sigma_{xz} = \sum_{\lambda} \frac{EC^{(\lambda)}}{2(1+\nu)} \lambda r^{\lambda-1} \sin[(\lambda-1)\theta]$$
 & $\sigma_{yz} = \sum_{\lambda} \frac{EC^{(\lambda)}}{2(1+\nu)} \lambda r^{\lambda-1} \cos[(\lambda-1)\theta]$

- Constraints
 - Displacement field should be finite $\implies \lambda > 0$
 - Crack is stress free: $\cos[\pm(\lambda 1)\pi] \implies \lambda = n/2$, with n = 1, 2, ... since $\lambda > 0$

$$\int \sigma_{xz} = -\frac{EC}{4(1+\nu)\sqrt{r}} \sin\frac{\theta}{2} + M(\theta)r^{0} + N(\theta)\sqrt{r} \dots$$

$$\sigma_{yz} = \frac{EC}{4(1+\nu)\sqrt{r}} \cos\frac{\theta}{2} + O(\theta)r^{0} + P(\theta)\sqrt{r} \dots$$

$$- \text{SIF:} \quad K_{III} = \lim_{r \to 0} \left(\sqrt{2\pi r} \sigma_{yz}^{\text{mode III}}|_{\theta=0}\right) = \frac{CE}{2(1+\nu)}\sqrt{\frac{\pi}{2}}$$

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- Application to mode III (shearing) (4)
 - Asymptotic fields











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Asymptotic solution

• Summary in 2D



- Principle of superposition holds as linear responses have been assumed
 - K_i depends on
 - The geometry and
 - The loading
 - \boldsymbol{u}, σ can be added for
 - \neq modes: $\boldsymbol{u} = \boldsymbol{u}^{\text{mode I}} + \boldsymbol{u}^{\text{mode II}}, \sigma = \sigma^{\text{mode I}} + \sigma^{\text{mode II}}$
 - \neq loadings $\boldsymbol{u} = \boldsymbol{u}^{\text{loading 1}} + \boldsymbol{u}^{\text{loading 1}}, \boldsymbol{\sigma} = \boldsymbol{\sigma}^{\text{loading 1}} + \boldsymbol{\sigma}^{\text{loading 2}}$
 - K_i can be added
 - For \neq loadings of the same mode $K_i = K_i^{\text{loading 1}} + K_i^{\text{loading 2}}$
 - But since **f** and **g** depend on the mode $K \neq K_{\text{mode }1} + K_{\text{mode }2}$







• 1957, Irwin, new failure criterion

- $\sigma_{max} \rightarrow \infty \implies \sigma \text{ is irrelevant}$
- Compare the SIFs (dependent on loading and geometry) to a new material property: the toughness
 - If $K_i = K_{iC} \implies$ crack growth
 - Toughness (ténacité) K_{Ic}
 - Steel, Al, ... : see figures
 - Concrete: 0.2 1.4 MPa m^{1/2}





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- Measuring K_{Ic}
 - Done by strictly following the ASTM E399 procedure
 - Specimen
 - Normalized, e.g. Single Notched Bend (SENB) •
 - Plane strain constraint (thick enough) ٠
 - conservative (see next slide)
 - Specimen machined with a V-notch
 - Crack initiation
 - Cyclic loading to initiate a fatigue crack •
 - Crack length from compliance •
 - Crack Mouth Opening Displacement
 - (CMOD=v) measured with a clipped gauge
 - Calibrated using FEM











- Measuring K_{ic} (2)
 - Done by strictly following the ASTM E399 procedure
 - Toughness test
 - Calibrated P, δ recording equipment
 - Crack Mouth Opening Displacement (CMOD=v) measured with a clipped gauge
 - P_c is obtained on P v curves
 - Either the 95% offset value or
 - The maximal value reached before
 - K_{Ic} is deduced from P_c using

$$K_I = \frac{PL}{tW^{\frac{3}{2}}} f\left(\frac{a}{W}\right)$$

- f(a/W) depends on the test (SENB, ...)
- f(a/W) calibrated using FEM etc, in the norm
- Check the constraint once you have K_{Ic}
 - Plane strain constraint (thick enough)









3D problems

- Only 2D solutions have been considered but a real crack is clearly 3D
 - At any point of the crack line _
 - A local referential can be defined ٠
 - Since the asymptotic solutions hold for ٠
 - $r \rightarrow 0$, at this distance the crack line seems
 - straight, and the problem is locally 2D
 - The crack tip field can be broken into ٠ 3 2D problems (3 2D modes)









3D problems

• 3D effects

- Near the border of a specimen the problem is plane σ , while it is plane ϵ near the center
 - → at the center there is a triaxial state
 - The SIF is larger at the center as no lateral deformations are possible (see next lecture)
 - 2 consequences
 - The front will first propagate at the center
 - The toughness decreases with the thickness
 - » Crude approximation

$$K_C(t) \simeq K_C(t \to \infty) \left[1 + \frac{1.4}{t^2} \left(\frac{K_C(t \to \infty)}{\sigma_p^0} \right)^4 \right]^{\frac{1}{2}}$$

» Later on we will see how to evaluate this effect

- The practical toughness K_c is the plane strain one







- Evaluation of the stress Intensity Factor (SIF)
 - Analytical (crack 2a in an infinite plane)



Geometry

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Crack length





References

- Lecture notes
 - Lecture Notes on Fracture Mechanics, Alan T. Zehnder, Cornell University, Ithaca, <u>http://hdl.handle.net/1813/3075</u>
 - Fracture Mechanics Online Class, L. Noels, ULg, <u>http://www.ltas-</u> <u>cm3.ulg.ac.be/FractureMechanics</u>
- Book
 - Fracture Mechanics: Fundamentals and applications, D. T. Anderson. CRC press, 1991.
 - S. Suresh, Fatigue of Materials, Cambridge University Press, 2001







Annex 1: Plate with a hole

Cylindrical coordinates σ_{∞} $\left(egin{array}{ccc} \boldsymbol{\sigma}_{rr} & \boldsymbol{\sigma}_{r heta} \ \boldsymbol{\sigma}_{r heta} & \boldsymbol{\sigma}_{ heta heta} \end{array}
ight) = \left(egin{array}{ccc} \cos heta & \sin heta \ -\sin heta & \cos heta \end{array}
ight)$ σ_{yy} σ_{yy} $\left(egin{array}{cc} oldsymbol{\sigma}_{xx} & oldsymbol{\sigma}_{xy} \\ oldsymbol{\sigma}_{xy} & oldsymbol{\sigma}_{yy} \end{array}
ight) \left(egin{array}{cc} \cos heta & \sin heta \\ -\sin heta & \cos heta \end{array}
ight)^T$ $h \rightarrow \infty$ х Boundary conditions σ_{∞} σ_{∞} • At r = a $\boldsymbol{\sigma}_{rr} (r=a,\theta) = \boldsymbol{\sigma}_{r\theta} (r=a,\theta) = 0$ • At $r = b \rightarrow \infty$: only $\sigma_{vv} = \sigma_{\infty}$ is non zero $\begin{cases} \boldsymbol{\sigma}_{rr} \left(r = b, \theta \right) = \sigma_{\infty} \sin^{2} \theta \neq \frac{1}{2} \sigma_{\infty} - \frac{1}{2} \sigma_{\infty} \cos 2\theta \\ \boldsymbol{\sigma}_{r\theta} \left(r = b, \theta \right) = \sigma_{\infty} \cos \theta \sin \theta \neq \frac{1}{2} \sigma_{\infty} \sin 2\theta \end{cases}$ Load case 2 Load case 1 dependent on $\cos 2\theta$ independent (symmetry with from θ respect to x)







Annex 1: Plate with a hole

• Cylindrical coordinates (2)

- Bi-harmonic equation $\nabla^2 \nabla^2 \Phi = 0$

with
$$\begin{pmatrix} \partial_x \\ \partial_y \end{pmatrix} = \begin{pmatrix} \cos\theta & -\frac{\sin\theta}{r} \\ \sin\theta & \frac{\cos\theta}{r} \end{pmatrix} \begin{pmatrix} \partial_r \\ \partial_\theta \end{pmatrix}$$

 $\implies \left(\partial_{rr}^2 + \frac{1}{r} \partial_r + \frac{1}{r^2} \partial_{\theta\theta}^2 \right) \left(\Phi_{,rr} + \frac{1}{r} \Phi_{,r} + \frac{1}{r^2} \Phi_{,\theta\theta} \right) = 0$

Stresses from Airy function

$$\begin{cases} \boldsymbol{\sigma}_{rr} = \boldsymbol{\sigma}_{xx}\cos^{2}\theta + \boldsymbol{\sigma}_{yy}\sin^{2}\theta + 2\boldsymbol{\sigma}_{xy}\sin\theta\cos\theta\\ \boldsymbol{\sigma}_{\theta\theta} = \boldsymbol{\sigma}_{yy}\cos^{2}\theta + \boldsymbol{\sigma}_{xx}\sin^{2}\theta - 2\boldsymbol{\sigma}_{xy}\sin\theta\cos\theta\\ \boldsymbol{\sigma}_{r\theta} = (\boldsymbol{\sigma}_{yy} - \boldsymbol{\sigma}_{xx})\sin\theta\cos\theta + \boldsymbol{\sigma}_{xy}\left(\cos^{2}\theta - \sin^{2}\theta\right) \end{cases}$$

with $\sigma_{\alpha\beta} = -\Phi_{,\alpha\beta} + \delta_{\alpha\beta}\Phi_{,\gamma\gamma} \implies \sigma_{xx} = \Phi_{,yy}$, $\sigma_{yy} = \Phi_{,xx}$ & $\sigma_{xy} = -\Phi_{,xy}$ • Eventually, after substitutions $\begin{cases} \sigma_{rr} = \frac{1}{r}\Phi_{,r} + \frac{1}{r^2}\Phi_{,\theta\theta} \\ \sigma_{\theta\theta} = \Phi_{,rr} \\ \sigma_{r\theta} = -\frac{1}{r}\Phi_{,r\theta} + \frac{1}{r^2}\Phi_{,\theta} \end{cases}$





 $b \rightarrow \infty$

 $|\sigma_{\infty}|$

• Load case 1

_

- Independent from $\theta \implies \Phi(r)$ • Bi-harmonic equation $\implies \left(\partial_{rr}^2 + \frac{1}{r}\partial_r\right)\left(\Phi_{,rr} + \frac{1}{r}\Phi_{,r}\right) = 0$
 - General solution: $\Phi = C_1 \ln r + C_2 r^2 \ln r + C_3 r^2 + C_4$

• Stresses:
$$\begin{cases} \boldsymbol{\sigma}_{rr} = \frac{1}{r} \Phi_{,r} = \frac{C_1}{r^2} + C_2 \left(1 + 2\ln r\right) + 2C_3 \\ \boldsymbol{\sigma}_{\theta\theta} = \Phi_{,rr} = -\frac{C_1}{r^2} + C_2 \left(3 + 2\ln r\right) + 2C_3 \end{cases}$$

$$\implies \frac{C_1}{a^2} + C_2 \left(1 + 2\ln a\right) + 2C_3 = 0 \& \lim_{b \to \infty} \left(\frac{C_1}{b^2} + C_2 \left(1 + 2\ln b\right) + 2C_3\right) = \frac{1}{2}\sigma_{\infty}$$
$$\implies C_1 = -\frac{a^2}{2}\sigma_{\infty} , \quad C_2 = 0 \& C_3 = \frac{1}{4}\sigma_{\infty}$$
$$\bullet \text{ Solution: } \sigma_{rr} = \frac{1}{2}\sigma_{\infty} \left(1 - \frac{a^2}{r^2}\right), \quad \sigma_{\theta\theta} = \frac{1}{2}\sigma_{\infty} \left(\frac{a^2}{r^2} + 1\right) \& \sigma_{r\theta} = 0$$





• Load case 2

•

- Dependent on $\cos 2\theta \implies \Phi(r, \cos 2\theta) = g(r) \cos 2\theta$

• Bi-harmonic equation

$$\left(\partial_{rr}^{2} + \frac{1}{r}\partial_{r} + \frac{1}{r^{2}}\partial_{\theta\theta}^{2}\right)\left[\left(g'' + \frac{g'}{r} - \frac{4g}{r^{2}}\right)\cos 2\theta\right] = 0$$

$$\implies \left(g'''' + \frac{2g'''}{r} - \frac{9g''}{r^{2}} + \frac{9g'}{r^{3}}\right)\cos 2\theta = 0$$

General solution:
$$\Phi = \left(C_{1}r^{4} + C_{2}r^{2} + \frac{C_{3}}{r^{2}} + C_{4}\right)\cos 2\theta$$

• Stresses:
$$\begin{cases} \boldsymbol{\sigma}_{rr} = \left(\frac{g'}{r} - \frac{4g}{r^2}\right)\cos 2\theta = \left(-2C_2 - \frac{6C_3}{r^4} - \frac{4C_4}{r^2}\right)\cos 2\theta \\ \boldsymbol{\sigma}_{\theta\theta} = g''\cos 2\theta = \left(12C_1r^2 + 2C_2 + \frac{6C_3}{r^4}\right)\cos 2\theta \\ \boldsymbol{\sigma}_{r\theta} = \left(\frac{2g'}{r} - \frac{2g}{r^2}\right)\sin 2\theta = \left(6C_1r^2 + 2C_2 - \frac{6C_3}{r^4} - \frac{2C_4}{r^2}\right)\sin 2\theta \end{cases}$$



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- Load case 2 (2)
 - Stresses

$$\begin{cases} \boldsymbol{\sigma}_{rr} = \left(\frac{g'}{r} - \frac{4g}{r^2}\right)\cos 2\theta = \left(-2C_2 - \frac{6C_3}{r^4} - \frac{4C_4}{r^2}\right)\cos 2\theta \\ \boldsymbol{\sigma}_{\theta\theta} = g''\cos 2\theta = \left(12C_1r^2 + 2C_2 + \frac{6C_3}{r^4}\right)\cos 2\theta \\ \boldsymbol{\sigma}_{r\theta} = \left(\frac{2g'}{r} - \frac{2g}{r^2}\right)\sin 2\theta = \left(6C_1r^2 + 2C_2 - \frac{6C_3}{r^4} - \frac{2C_4}{r^2}\right)\sin 2\theta \end{cases}$$

– Boundary conditions:

$$\begin{cases} \boldsymbol{\sigma}_{rr} \left(r=a,\theta\right) = \boldsymbol{\sigma}_{r\theta} \left(r=a,\theta\right) = 0\\ \boldsymbol{\sigma}_{rr} \left(r=b,\theta\right) = -\frac{1}{2}\sigma_{\infty}\cos 2\theta\\ \boldsymbol{\sigma}_{r\theta} \left(r=b,\theta\right) = \frac{1}{2}\sigma_{\infty}\sin 2\theta\\ \implies C_{1} = 0 \quad , C_{2} = \frac{\sigma_{\infty}}{4}, C_{3} = \frac{\sigma_{\infty}a^{4}}{4} \quad \& \quad C_{4} = -\frac{\sigma_{\infty}a^{2}}{2} \end{cases}$$





• Load case 2 (2)

– Solution:

$$\sigma_{rr} = \frac{\sigma_{\infty}}{2} \left(\frac{4a^2}{r^2} - 1 - \frac{3a^4}{r^4} \right) \cos 2\theta$$
$$\sigma_{\theta\theta} = \frac{\sigma_{\infty}}{2} \left(1 + \frac{3a^4}{r^4} \right) \cos 2\theta$$
$$\sigma_{r\theta} = \frac{\sigma_{\infty}}{2} \left(1 - \frac{3a^4}{r^4} + \frac{2a^2}{r^2} \right) \sin 2\theta$$







• Superposition of case 1 and 2

$$- \text{ Stresses:} \begin{cases} \sigma_{rr} = \frac{\sigma_{\infty}}{2} \left[\left(\frac{4a^2}{r^2} - 1 - \frac{3a^4}{r^4} \right) \cos 2\theta + 1 - \frac{a^2}{r^2} \right] \\ \sigma_{\theta\theta} = \frac{\sigma_{\infty}}{2} \left[\left(1 + \frac{3a^4}{r^4} \right) \cos 2\theta + \frac{a^2}{r^2} + 1 \right] \\ \sigma_{r\theta} = \frac{\sigma_{\infty}}{2} \left(1 - \frac{3a^4}{r^4} + \frac{2a^2}{r^2} \right) \sin 2\theta \\ - \text{ For } \theta = 0; \ \sigma_{yy} = \sigma_{\theta\theta} \qquad \qquad 3 \\ \implies \sigma_{yy} \left(x, y = 0 \right) = \frac{\sigma_{\infty}}{2} \left(2 + \frac{3a^4}{x^4} + \frac{a^2}{x^2} \right) \begin{array}{c} 2.5 \\ 2 \\ - \end{array} \\ \text{ Stress intensity factor: } K_{\text{circle}} = 3 \qquad 1.5 \\ 1 \\ 0.5 \\ 0 \\ 0 \qquad 2 \qquad 4 \qquad 6 \end{cases}$$



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- Displacements for linear and elastic materials
 - Complex form: $\boldsymbol{u} = \boldsymbol{u}_x \left(x, \, y \right) + i \boldsymbol{u}_y \left(x, \, y \right) = \boldsymbol{u} \left(\zeta, \, \bar{\zeta} \right)$
 - In terms of strains:

$$\boldsymbol{u}_{,\bar{\zeta}} = (\boldsymbol{u}_{x,x} + i\boldsymbol{u}_{y,x}) \,\partial_{\bar{\zeta}}x + (\boldsymbol{u}_{x,y} + i\boldsymbol{u}_{y,y}) \,\partial_{\bar{\zeta}}y = \frac{1}{2} \left(\boldsymbol{u}_{x,x} + i\boldsymbol{u}_{y,x} + i\boldsymbol{u}_{x,y} - \boldsymbol{u}_{y,y} \right)$$

$$\implies \boldsymbol{u}_{,\bar{\zeta}} = \frac{1}{2} \left(\boldsymbol{\varepsilon}_{xx} - \boldsymbol{\varepsilon}_{yy} \right) + i\boldsymbol{\varepsilon}_{xy}$$

Hooke's law (Plane σ or ε): _

$$\boldsymbol{u}_{,\bar{\zeta}} = \frac{1+\nu}{2E} \left(\boldsymbol{\sigma}_{xx} - \boldsymbol{\sigma}_{yy} + 2i\boldsymbol{\sigma}_{xy} \right) = -\frac{1+\nu}{E} \left(\zeta \bar{\Omega}'' + \bar{\omega}'' \right)$$
$$\implies \boldsymbol{u} = -\frac{1+\nu}{E} \left(\zeta \bar{\Omega}' + \bar{\omega}' + \mu \left(\zeta \right) \right)$$

With $\mu(\zeta)$ such that the plane σ or ε condition is satisfied: •





- Displacements for linear and elastic materials (2)
 - Out-of-plane state effect

$$\varepsilon_{\gamma\gamma} = -\frac{1+\nu}{E} \left(\bar{\Omega}' + \mu' + \Omega' + \bar{\mu}' \right)$$

• Plane σ : $\varepsilon_{\gamma\gamma} = \frac{1-\nu}{E} \sigma_{\gamma\gamma} = \frac{2(1-\nu)}{E} \left(\Omega' + \bar{\Omega}' \right)$
 $\implies \mu(\zeta) = -\frac{3-\nu}{1+\nu} \Omega = -\kappa \Omega(\zeta) \text{ with } \kappa = \frac{3-\nu}{1+\nu}$
• Plane ε : $\varepsilon_{\gamma\gamma} = \frac{(1+\nu)(1-2\nu)}{E} \sigma_{\gamma\gamma} = \frac{2(1+\nu)(1-2\nu)}{E} \left(\Omega' + \bar{\Omega}' \right)$
 $\implies \mu(\zeta) = -(3-4\nu) \Omega = -\kappa \Omega(\zeta) \text{ with } \kappa = 3 - 4\nu$
For both plane states: $u = -\frac{1+\nu}{E} \left(\zeta \bar{\Omega}' + \bar{\omega}' - \kappa \Omega(\zeta) \right)$



